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E. Love, W.J. Rider, G. Scovazzi
Sandia National Laboratories
P.O. Box 5800, MS-0378
Albuquerque, NM 87185-0378

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Outline of Presentation

- Introduction and motivation
- Continuum equations
 - Kinematics
 - Balance Laws
- Artificial viscosity
 - Hyperbolic PDE concepts
 - Flux limiting
 - Numerical simulations
- Hyperviscosity
 - Algorithm
 - Numerical simulations
- Concluding remarks



In search of a better artificial viscosity

- Typical artificial viscosity methods for Lagrangian hydrodynamic calculations are only first-order accurate.
- The shock-capturing viscosity is active in compression, regardless of whether the compression is adiabatic or a shock.
- Ideally the viscosity should vanish (go to zero) if the fluid flow is smooth (adiabatic/isentropic).
- The goal is to construct a second-order accurate artificial viscosity method, one which can "tell the difference" between shocked and smooth flows.

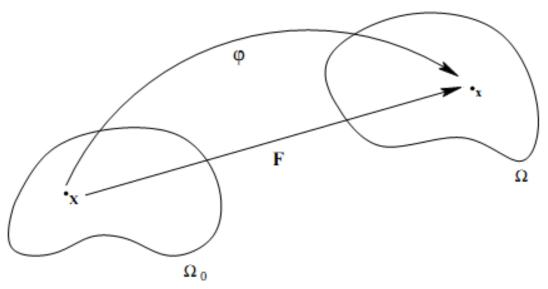


Previous Work

- J. VonNeumann and R.D. Richtmyer, "A Method for the Numerical Calculation of Hydrodynamic Shocks", Journal of Applied Physics, 21(3), March 1950, 232-237.
- [2] Mark L. Wilkins, "Use of artificial viscosity in multidimensional fluid dynamic calculations", Journal of Computational Physics, 36(3), July 1980, 281-303.
- [3] Tz.V. Kolev and R.N. Rieben, "A tensor artificial viscosity using a finite element approach", Journal of Computational Physics, 228(1), December 2009, 8336-8366.
- [4] Culbert B. Laney, Computational Gasdynamics, Cambridge University Press, 1998.



Basic kinematics underlying hyperbolic flow



$$\varphi:\Omega_0\times[0,T]\to\Omega\subset\mathbb{R}^3$$

$$\mathbf{F} := D\boldsymbol{\varphi} = \mathrm{GRAD}[\boldsymbol{\varphi}]$$

$$J = \det[\mathbf{F}]$$

$$\mathbf{v}=\dot{oldsymbol{arphi}}$$

$$\operatorname{grad}[\cdot] = \mathbf{F}^{-T}\operatorname{GRAD}[\cdot] \Longrightarrow \operatorname{grad}[\mathbf{v}] = \dot{\mathbf{F}}\mathbf{F}^{-1}$$



Integral Lagrangian form of the conservation laws

Conservation of mass

$$\rho_0 - \rho J = 0$$

Conservation of linear momentum

$$\int_{\Omega_0} \boldsymbol{\eta} \bullet \rho_0 \dot{\mathbf{v}} \, d\Omega_0 + \int_{\Omega} \operatorname{grad}^s[\boldsymbol{\eta}] \bullet (-p\mathbf{I} + \boldsymbol{\sigma}_{art}) \, d\Omega = 0 \quad \forall \boldsymbol{\eta}$$

Conservation of energy

$$\int_{\Omega_0} \phi \cdot \rho_0 \dot{\epsilon} \, d\Omega_0 - \int_{\Omega} \phi \cdot \operatorname{grad}^s[\mathbf{v}] \bullet (-p\mathbf{I} + \boldsymbol{\sigma}_{art}) \, d\Omega = 0 \quad \forall \phi$$

 This is a global system of hyperbolic conservation laws written in weak form.



Spatial discretization of hyperbolic conservation laws

Basic form

$$\partial_t \mathbf{u}^h + \partial_x^h \left[\mathbf{F}^h(\mathbf{u}^h) \right] = \mathbf{0}$$

- Decompose the numerical flux \mathbf{F}^h into high-order and low-order contributions. (Flux-corrected transport, self-adjusting hybrid schemes, TVD*,...)
- Self-adjusting hybrid scheme

$$\mathbf{F}^{h}(\mathbf{u}^{h}) = (1 - \theta)\mathbf{F}^{HO}(\mathbf{u}^{h}) + \theta\mathbf{F}^{LO}(\mathbf{u}^{h})$$

$$\theta_i = \frac{\|u_{i+1} - 2u_i + u_{i-1}\|}{\|u_{i+1} - u_i\| + \|u_i - u_{i-1}\|} \qquad 1-D$$

• The limiter θ_i looks like a normalized Laplacian.



Maybe one can use the velocity Laplacian to limit the artificial viscosity?

- A linear velocity field is smooth, does not represent a shocked flow, and also has zero Laplacian.
- Computation of the velocity Laplacian

$$(\nabla^{2}\mathbf{v}) = \nabla(\nabla \cdot \mathbf{v}) - \nabla \times (\nabla \times \mathbf{v}) = \operatorname{div}[\operatorname{grad}[\mathbf{v}]]$$
$$\int_{\Omega} \boldsymbol{\eta} \bullet (\nabla^{2}\mathbf{v}) \, d\Omega = -\int_{\Omega} \operatorname{grad}[\boldsymbol{\eta}] \bullet \operatorname{grad}[\mathbf{v}] \, d\Omega + \int_{\partial\Omega} \boldsymbol{\eta} \bullet \operatorname{grad}[\mathbf{v}] \, \mathbf{n} \, d\Gamma \quad \forall \boldsymbol{\eta}$$

Normalize using the triangle inequality

$$\theta_{A} = \frac{\left\| - \int_{\Omega_{A}} \operatorname{grad}[\mathbf{v}] \operatorname{grad}[N^{A}] + \int_{\partial \Omega_{A}} \operatorname{grad}[\mathbf{v}] N^{A} \mathbf{n} \right\|}{\int_{\Omega_{A}} \left\| \operatorname{grad}[\mathbf{v}] \operatorname{grad}[N^{A}] \right\| + \int_{\partial \Omega_{A}} \left\| \operatorname{grad}[\mathbf{v}] N^{A} \mathbf{n} \right\|} \leq 1$$



General structure of an improved artificial viscosity has several elements

- High-order "flux" is "zero artificial viscosity".
- Low-order "flux" is "standard artificial viscosity".
- Limited artificial viscosity if trace[d] < 0.0

$$\mathbf{d} = \operatorname{grad}^{s}[\mathbf{v}]$$

$$\boldsymbol{\sigma}_{art}^{LO} = \rho \left[c_{1} \ c \ h \ + \ c_{2} \ \|\operatorname{trace}[\mathbf{d}]\| \ h^{2} \ \right] \mathbf{d}$$

$$\boldsymbol{\sigma}_{art} = \theta \boldsymbol{\sigma}_{art}^{LO}$$

- If the velocity field is linear, then the artificial viscosity is zero on both the interior and the boundary of arbitrary unstructured meshes.
- Important to include boundary terms (red boxed terms on previous slide).

What happens asymptotically with mesh refinement?

- Assume the flow is smooth.
 - The standard non-limited artificial viscosity is O(h).
 - The limiter itself is O(h). In one dimension,

$$\theta_i = \frac{|u_{i+1} - 2u_i + u_{i-1}|}{|u_{i+1} - u_i| + |u_i - u_{i-1}|} = \frac{h|u''(x_i)|}{2|u'(x_i)|} + \mathcal{O}(h^3)$$

- When the artificial viscosity is multiplied by the limiter, the result is O(h²).
- The final limited viscosity is O(h²), and goes to zero one order faster than the standard artificial viscosity.
- Assume the flow is shocked, with a finite jump as $h\rightarrow 0$.

$$\{u_{i+1} = 1, u_i = 0, u_{i-1} = 0\} \implies \theta_i = 1$$

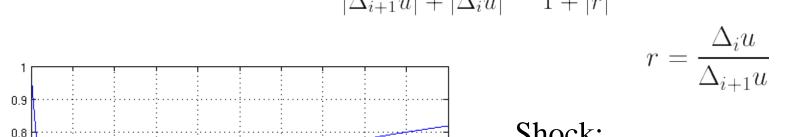
In this simple example situation, the limiter is one.



In general the limiter is a highly non-linear function of the discrete gradients

More generally,

$$\Delta_i u := (u_i - u_{i-1}) \Longrightarrow \theta_i = \frac{|\Delta_{i+1} u - \Delta_i u|}{|\Delta_{i+1} u| + |\Delta_i u|} = \frac{|1 - r|}{1 + |r|} \quad \text{(for } \Delta_{i+1} u > 0)$$



Shock:

$$\lim_{r \to 0} \theta_i = 1$$
$$\lim_{r \to \infty} \theta_i = 1$$

Smooth:

$$\lim_{r \to 1} \theta_i = 0$$



0.7

0.6

0.4

0.3

0.2

0.1

Φ 0.5

Details of the numerical implementation

- Use standard single-point integration (Q1/P0) fournode finite elements to discretize the weak form.
- Flanagan-Belytschko viscous hourglass control (scales linearly with sound speed) with parameter 0.05
- Second-order accurate (in time) predictor-corrector time integration algorithm.
- Artificial viscosity limiting based on Laplacian of velocity field.
- Gamma-law ideal gas equation-of-state.
- Constants $c_1 = 1.0$ and $c_2 = 1.5$.

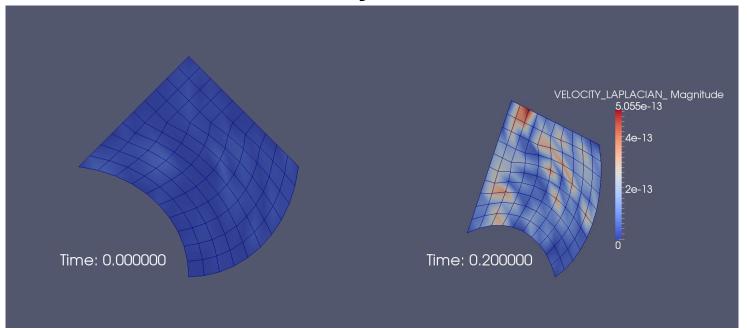


Zero Laplacian velocity field patch test

Linear velocity field

$$\mathbf{v} = (-2x_1 - x_2)\mathbf{e}_1 + (2x_1 - x_2)\mathbf{e}_2$$

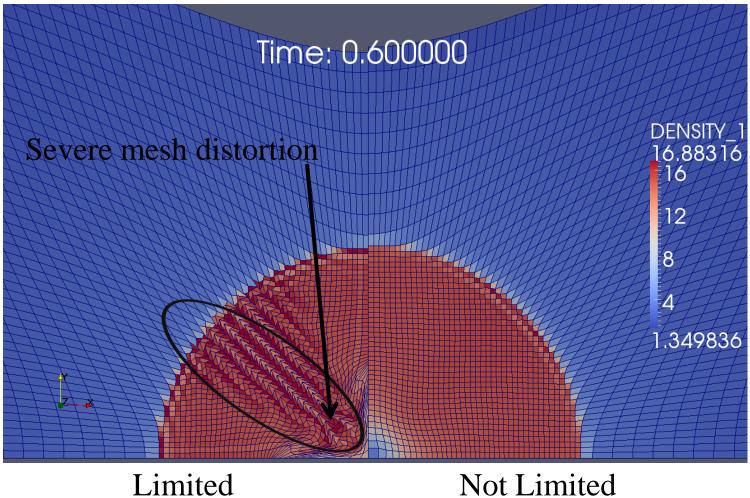
- Test on an initially distorted mesh.
- The velocity Laplacian is zero everywhere (test passes).
- Inclusion of the boundary terms is critical.





Numerical Simulations I

Noh Implosion Test





HyperViscosity

 Define d as the mean rate of deformation over a patch of elements.

$$\Omega_{patch} = \bigcup_{A=1}^{4} \operatorname{supp}(N^A)$$

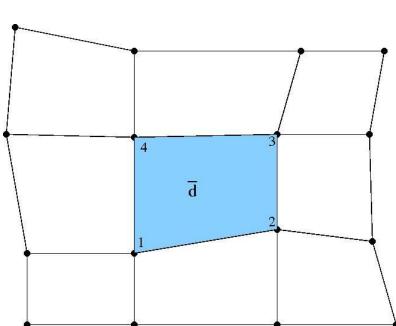
$$\bar{\mathbf{d}} = \frac{1}{\text{meas}(\Omega_{patch})} \int_{\Omega_{patch}} \mathbf{d} \, d\Omega$$

Add additional viscosity

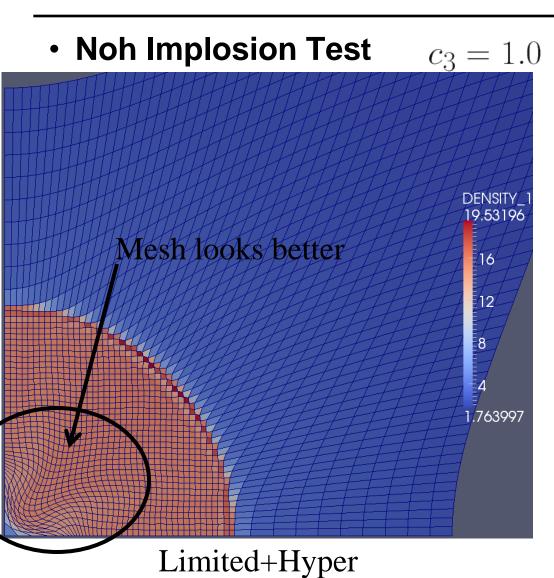
$$\boldsymbol{\sigma}_{hyper} = c_3 \left[\boldsymbol{\sigma}_{art}^{LO}(\mathbf{d}) - \boldsymbol{\sigma}_{art}^{LO}(\bar{\mathbf{d}}) \right]$$

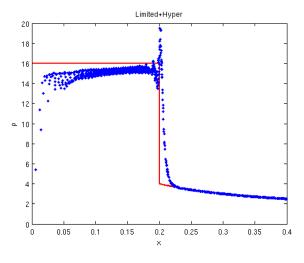
$$\sigma_{art} = \theta \sigma_{art}^{LO}(\mathbf{d}) + (1 - \theta) \sigma_{hyper}$$

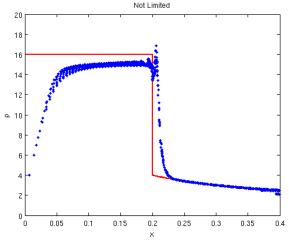
• The hyperviscosity also vanishes for a linear velocity field since in that case $\mathbf{d}=\bar{\mathbf{d}}$.



Numerical Simulations II

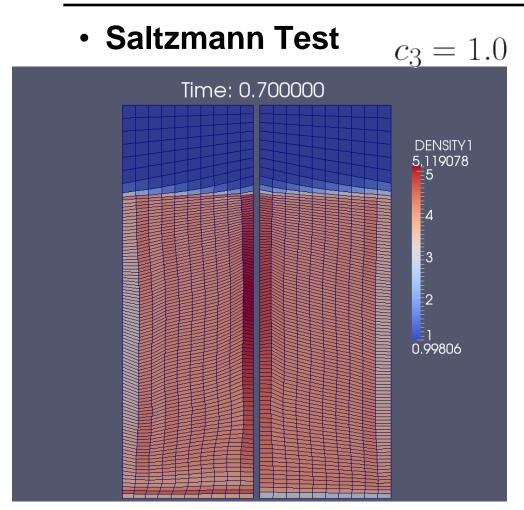


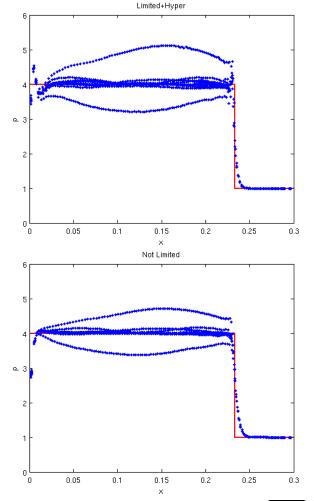






Numerical Simulations II





Limited+Hyper

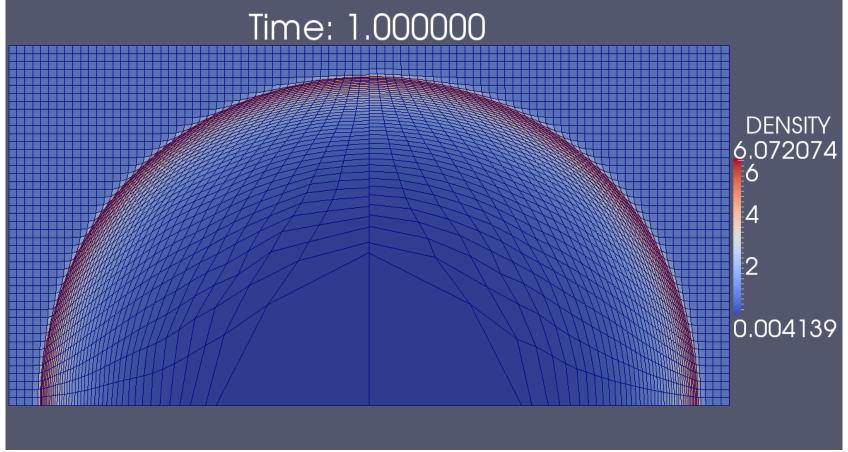
Not Limited



Numerical Simulations III

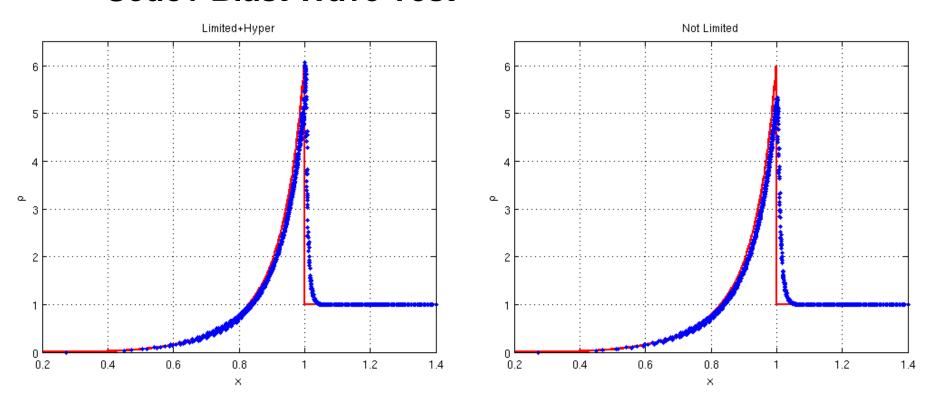
Sedov Blast Wave Test

$$c_3 = 1.0$$



Numerical Simulations III

Sedov Blast Wave Test



These results look promising...



Artificial viscosity limiting is a work in progress

- Limiter shows potential, but is strongly coupled to the hourglass control algorithm.
- What's the issue? Hypothesis...
 - Artificial viscosity causes heating (for an ideal gas), which increases sound speed, which increases hourglass control scaling?
 - Reducing artificial viscosity reduces heating (for an ideal gas), which reduces sound speed, which reduces hourglass control scaling?
- Good results for Eulerian (Lagrange+remap)
 simulations (where hourglassing is less of a problem).
- Much more work, including formal verification, is needed.